

# The Propulsion and manoeuvrability of double-ended ferries

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- 1. Design Requirements**
- 2. Preliminary Studies**
- 3. Hullform/Propulsion**
- 4. Manoeuvring**
- 5. Active Pass**
- 6. Conclusions**



# BCF- Requirements Super C-Class



## The Super C Program

370 Vehicles  
1500 Passengers  
21 knot service speed

2 Car Decks,  
single  
casing, *no  
ramps*

Dimensions  
Compatible with  
all Mainland  
Terminals

2 Passenger Decks:  
• Lounges/cafeteria/gift  
shop/snack bar/quiet  
lounge  
• 3 Vertical Zones  
arranged for shut-down  
• *Scope for expansion*

ABS Classification Standards  
Transport Canada Compliant



High Lift  
Rudders  
*-docking  
performance*

### Super C-Class

Diesel-Electric CPP constant  
speed propulsion plant:  
*Reliability, Life-cycle costs*

Double-Ended  
based on C-Class  
Configuration  
*- transit/access*

# Design Constraints

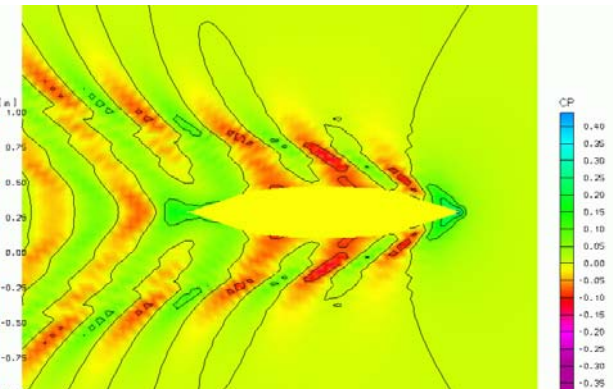
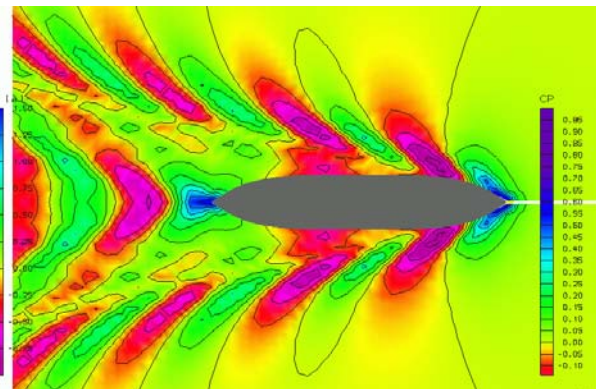
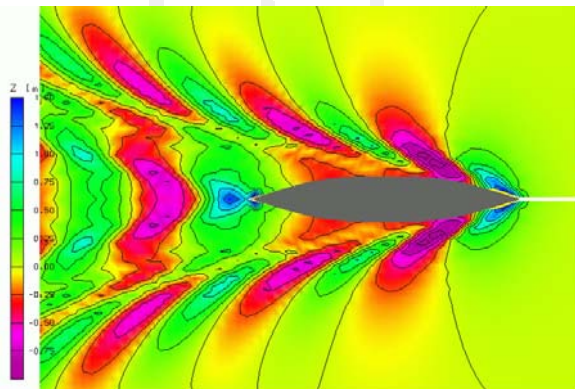
1. Deck WL fixed (Berth)
2. Lane- Capacity 370 AEQ, 1500 Pax
3.  $L < 160$  m,  $T > 5.75$  m und  $< 6.00$  m
4. Displacement ca. 9600 t incl. Growth Margin 500t
4. 21 kn w. 4 Prime Movers, 20kn w. 3 und 18kn w. 2  
incl. Fouling Margin
5. Diesel- Electric Plant
6. Turning Rate  $> 90$  Deg./min in Turning Circle
7. Acceleration and Crash Stop specified for 21 kn
8. 20/20 Overshoot less than 12.5 Deg (1/2 IMO)
9. Comfort ABS COMF+ (abt. DNV COMF 1), despite  
Propeller- „Tree- Class“ (1A- Super)

# Some Double Ender Concepts

**WEENTACHEE**  
L=140m v=21kn  
Bow-Stern Propeller

**SCHLESWIG-HOLSTEIN**  
L=142m v=22kn  
4 x Aquamaster CRP35

**SKEENA QUEEN**  
L=110m v=14kn  
4 x Niigata ZP 24



Contour of Wave Pattern

Pressure Distribution

Contour of Wave Pattern

Pressure Distribution

Contour of Wave Pattern

# Considerations for Propulsion

**Requirement: Diesel- Electric plant**

**Prime- Mover: 4 or 6 (Costs vs. Redundancy)**

<b>Propulsors : 2 (Bow- Stern) or</b>	<b>4 outside CL</b>
<b>Pods</b>	<b>Pods</b>
<b>FPPs</b>	<b>-</b>
<b>CPPs</b>	<b>-</b>

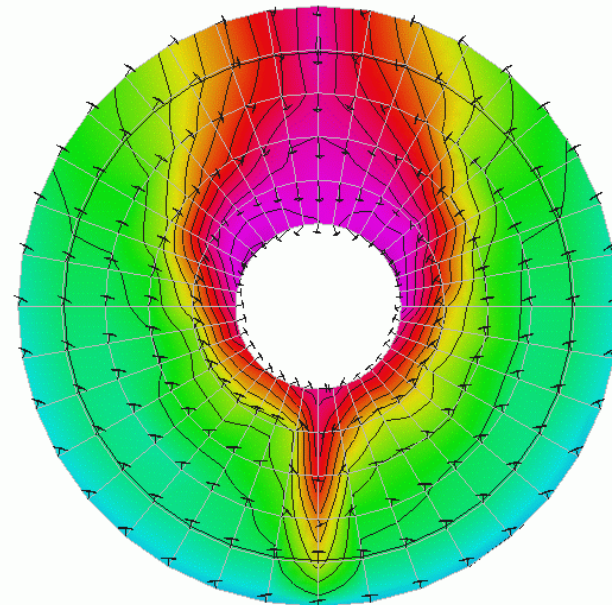
**6 Prime- Movers or 4 Pods not competitive (costs too high)**

**→ Solution must be Bow/Stern- Propulsor, 4 Prime Movers**

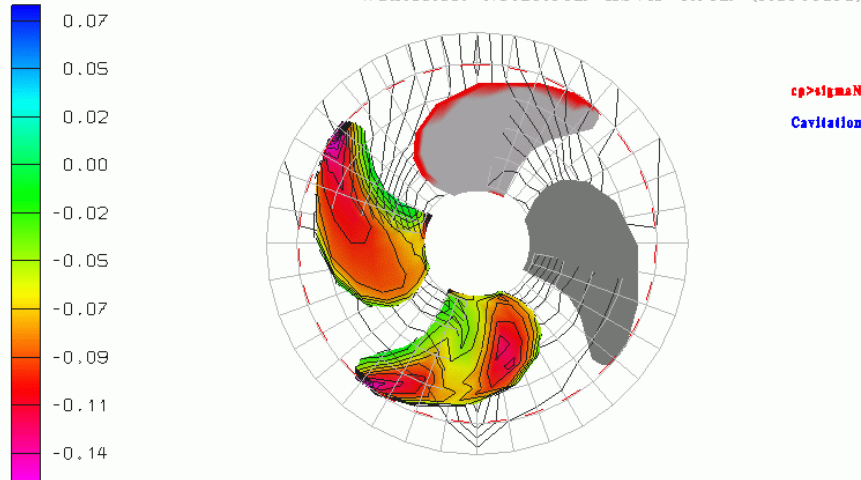
# Problem: Wakefield 2-Prop. Concept

Center Skeg essential for course keeping  
Disturbs wake field heavily !

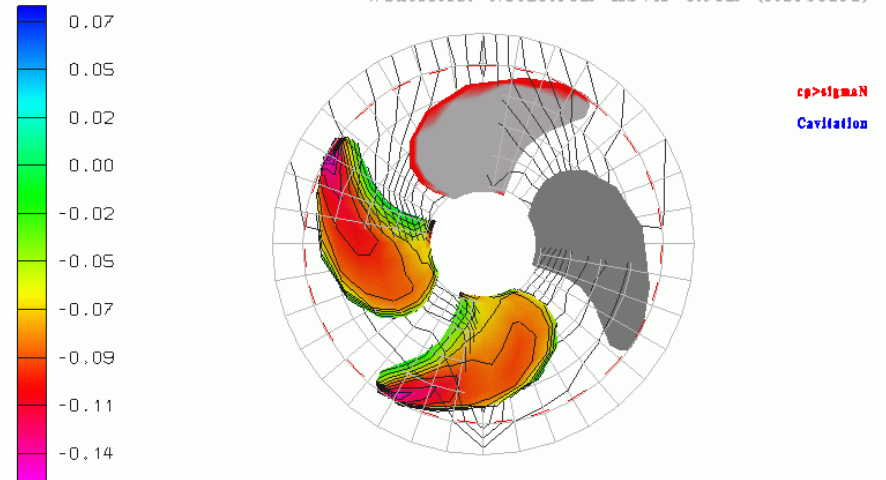
Example: Steering Fin DFDS  
1100 kW additional power due to  
Flow Separation at Blade.



WPM FINAL 2. RUNDE,  $\sigma_{nN} = 0.1882$   
Wakefield: Nachstrom HSVA 6.95m (starboard)



WPM FINAL 2. RUNDE,  $\sigma_{nN} = 0.1882$   
Wakefield: Nachstrom HSVA 6.95m (starboard)



HyKAT:  
Wake:

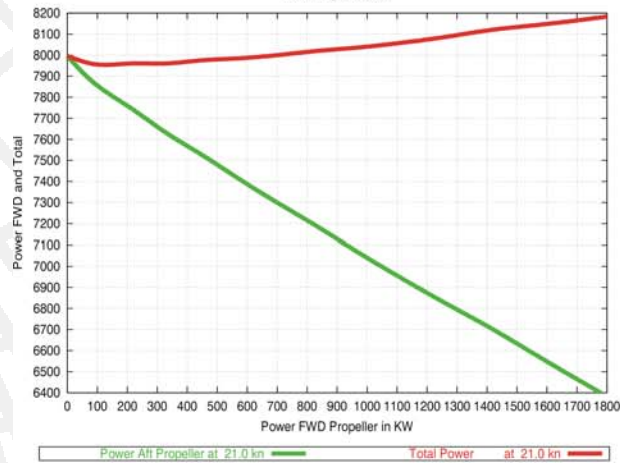
83.57Deg  
276.43Deg

HyKAT:  
Wake:

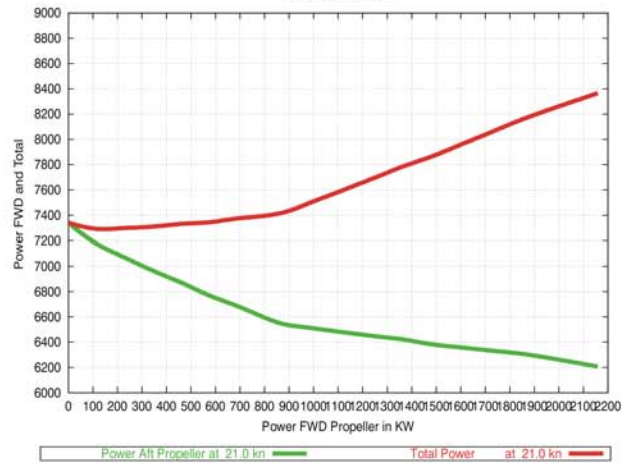
80.00Deg  
270.00Deg

# Power Sharing Considerations

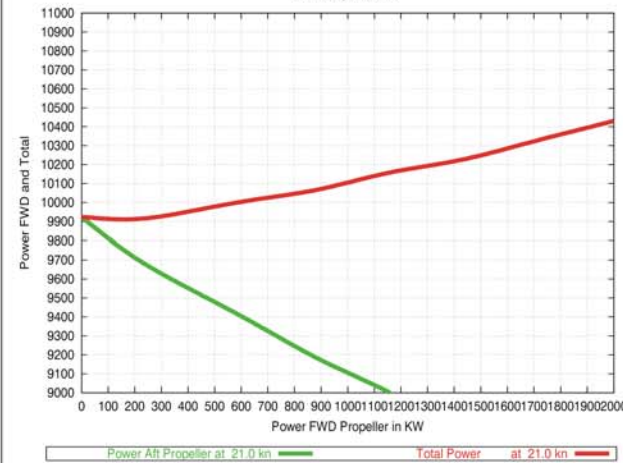
Power Optimization



Power Optimization



Power Optimization



## Queen of Oak Bay (BCF)

Propeller Diameter: 3.80m

P/D abt. 1.1 (MARIN)

Propeller Diameter: 3.80m

P/D abt. 1.67 (IOT)

## Super C-Class

Propeller Diameter: 5.0m

P/D abt. 1.4 (WPM)

**Power- Sharing makes sense only at high thrust loadings of stern propeller !**

**Stern Prop. To be designed sub- optimal: Larger Pitch !**

**Abt. 20% Power to be installed only for FWD Propulsor !**

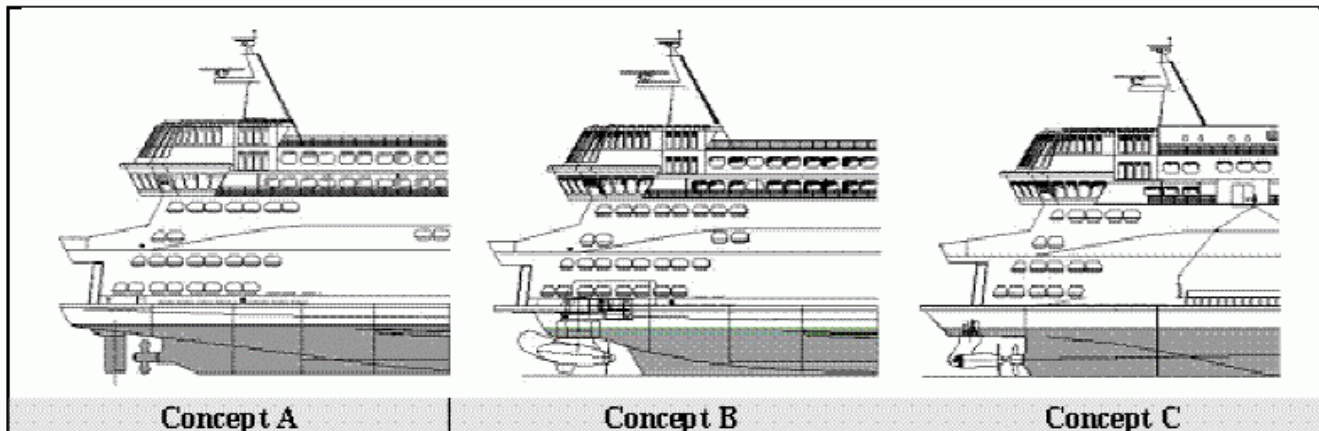
- 1. Extremely large stern prop required to avoid power sharing (large thrust deduction)**
- 2. Low resistance and thrust deduction required (system is sensitive w. respect to thrust loading)**
- 3. Pods not competitive as prop. torque too high**
- 4. Pods strut resistance for large prop. too high.**
- 5. Therefore Bow/Stern Prop as large CPP with constant revs.**



### 2. Hydrodynamic Model Tests – Double Ended Concepts

*Original C-Class tested in 1973; new program developed with several objectives:*

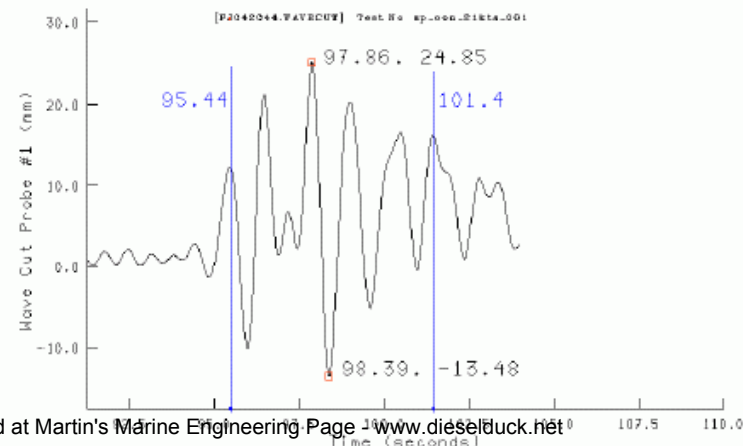
- I. Develop a baseline dataset for assessment of SY offers: hull form known to be extremely efficient baseline but no data at larger displacements
- II. Explore new propulsion concepts; application of podded systems to a double-ender; cost-benefit evaluation



### 2. Hydrodynamic Model Tests – Double Ended Concepts

*Program objectives (continued):*

- III. Investigate power sharing issues (bow/stern);
- IV. Investigate appendage drag issues; propeller feathering; parasitic drag of rudders
- V. "Wave cuts" for calibrating wake wash



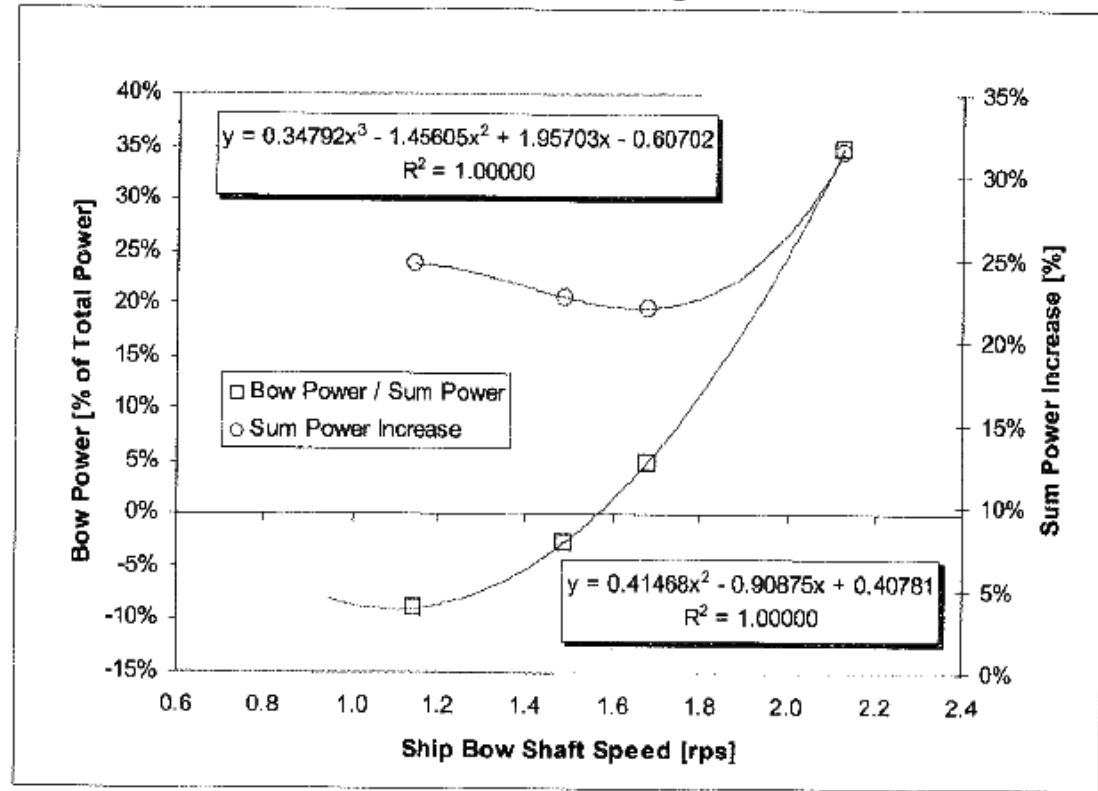
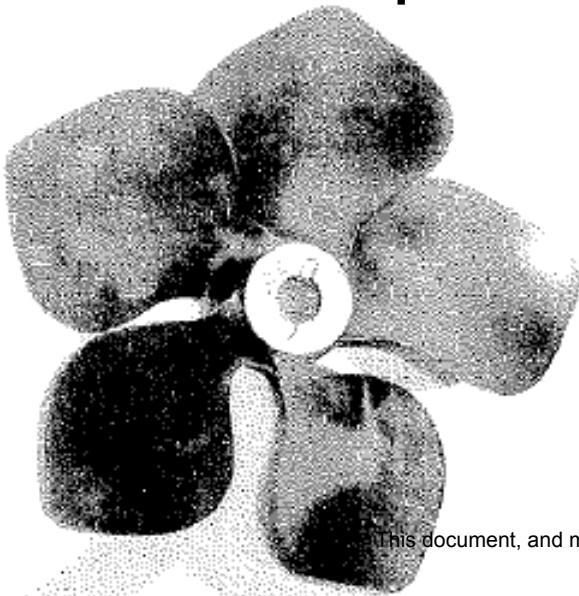
# Model Tests IOT Queen of Oak Bay

## Model Queen of Oak Bay /IOT)

Figure 5.1 : Power Sharing Results



Stock Propeller

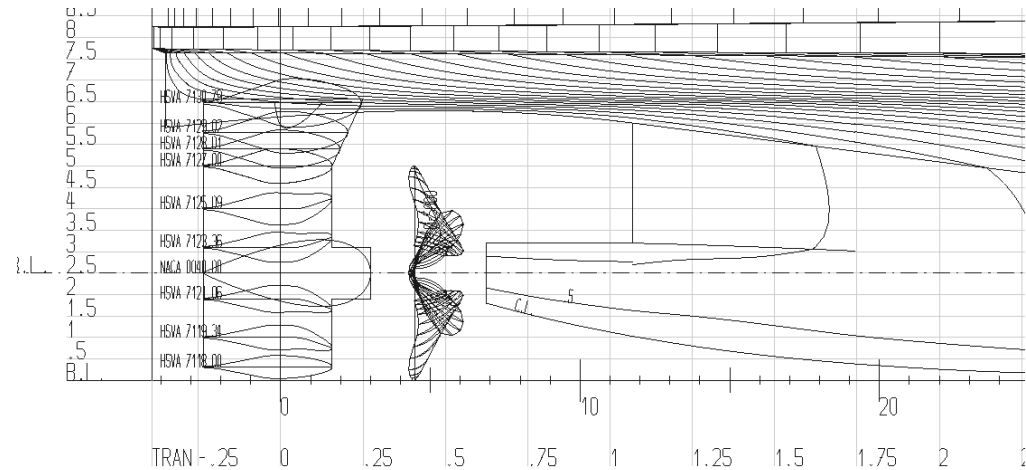


## Power Sharing Results

# Lessons learned from St. John's

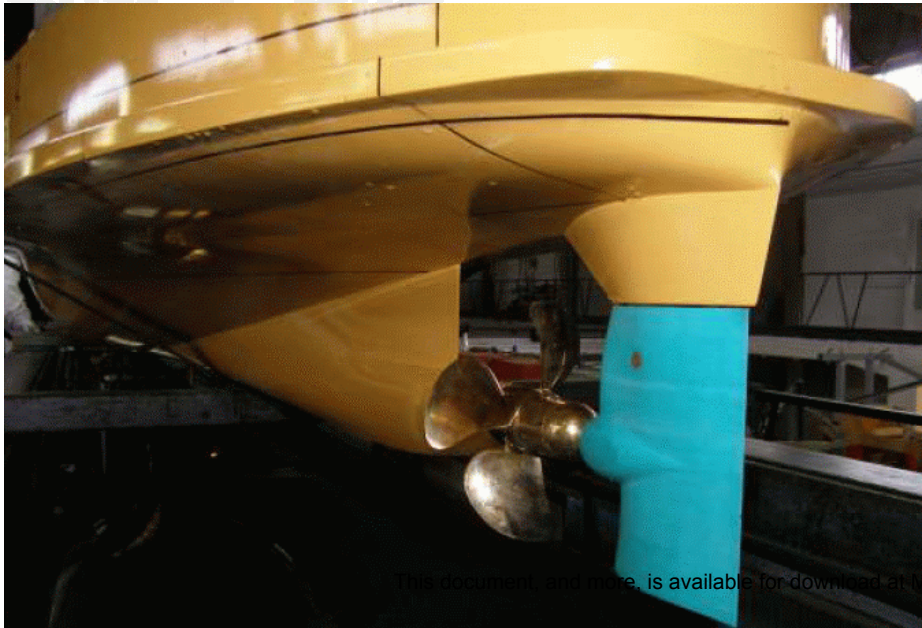
1. Existing Hull Form and Propulsion were not efficient (our point of view)
2. Our CFD Code was able to predict the wave pattern correctly despite numerical difficulties.
3. Our calculations for power sharing have shown to be sufficiently accurate.
4. The hull form should have extremely fine waterlines and buttocks at the ends.
5. The hull form should not disturb the wake but must provide sufficient course stability.
6. The appendage resistance of the bow rudder is significant and must be optimized.
7. Hull should dynamically trim by stern to improve course stability.

# Hull Form Design TUHH/FSG



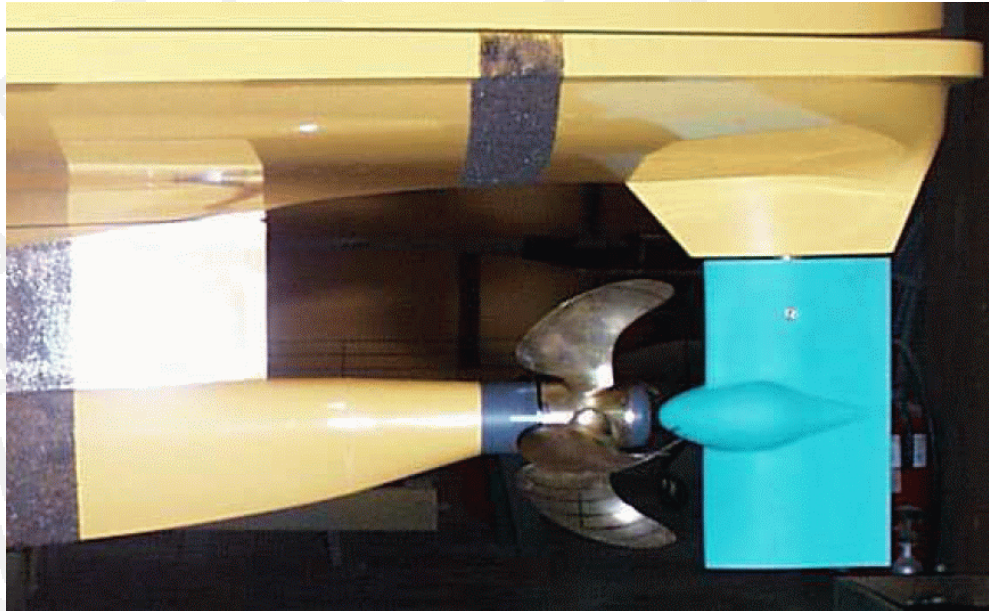
**Frame Plan**

**Appendages w. WPM CPP and FSG-TF- Rudder**



**HSVA- Model  
fitted with  
Final- Prop. and TF-Rudder**

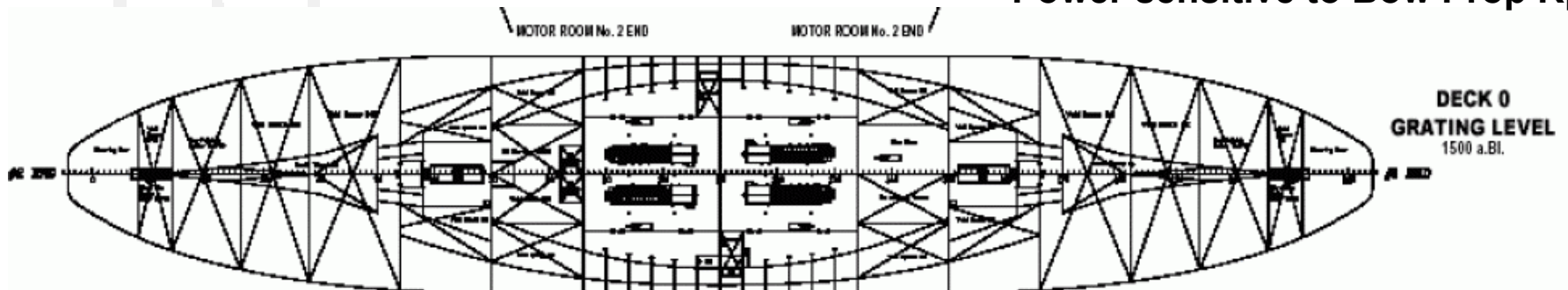
# Feathering Concept



**HSVA- Model  
fitted with  
Stock- Prop. and Stock-Rudder,  
Trailing Edge Feathering**

**Trailing Edge Feathering required due to:**

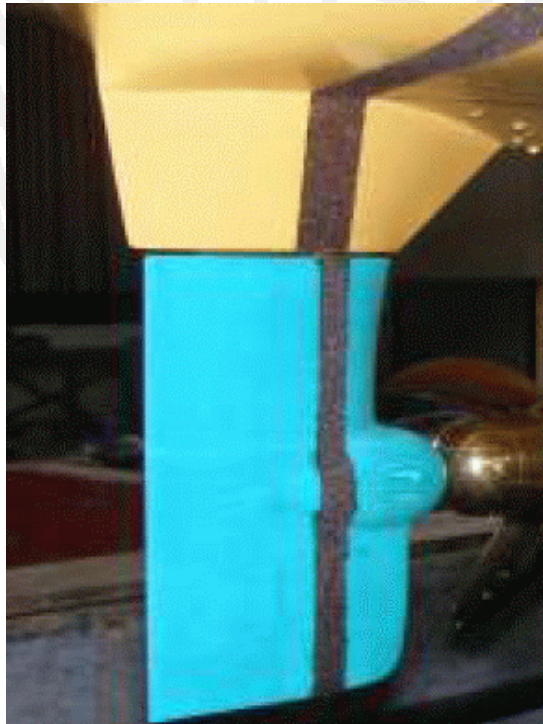
- Limited hub range (115 Deg)
- No clutch in shaft line
- Power sensitive to Bow Prop Rpm



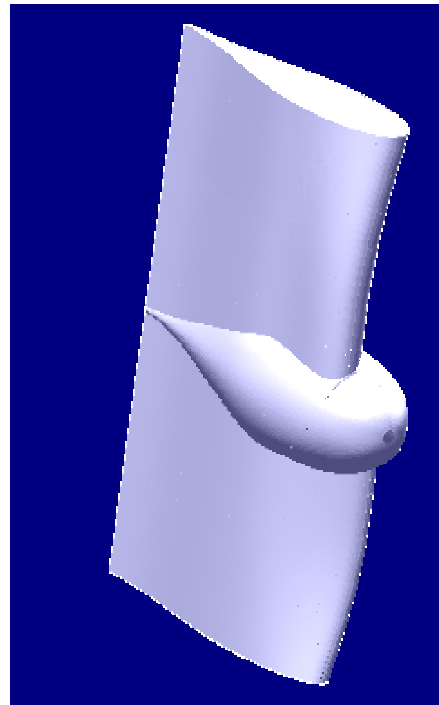
# TF- Rudder Optimization

**High- Lift FSG- TF Rudder designed wit bulb and profiles optimized for reverse condition.**

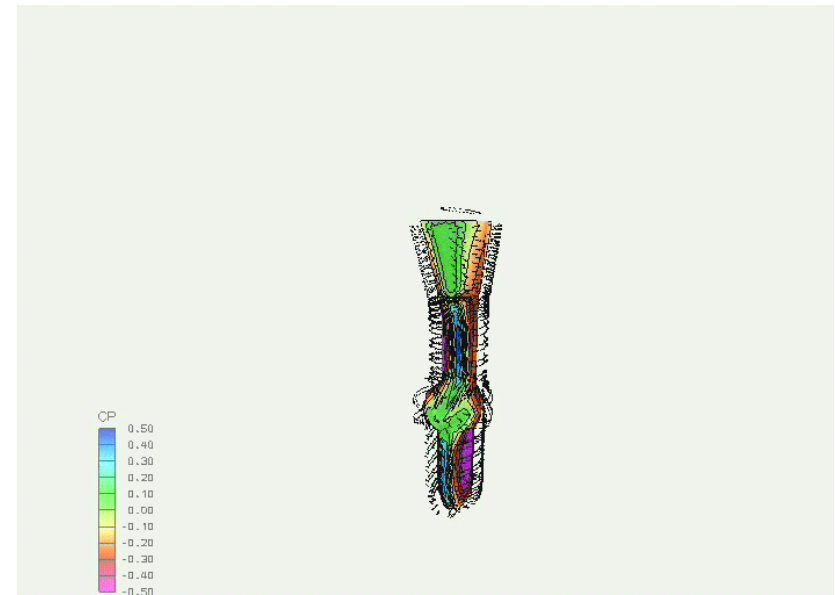
## Hardware



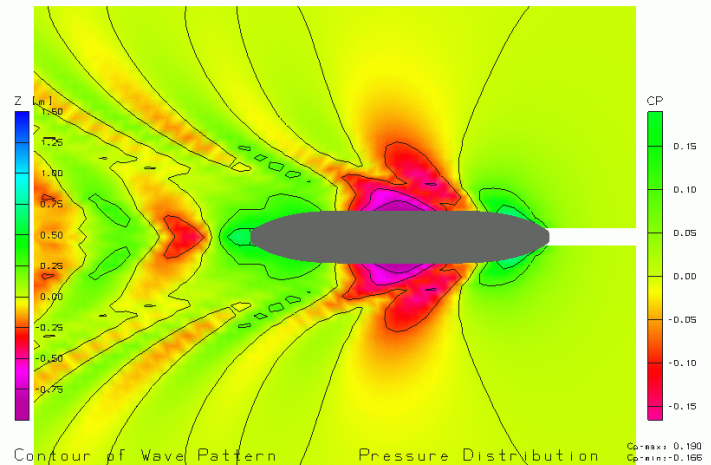
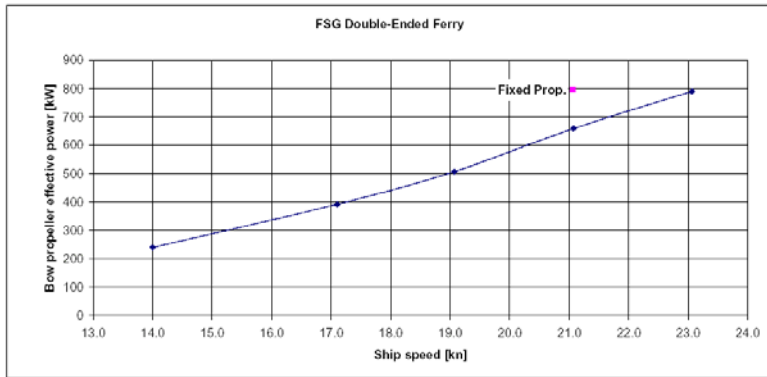
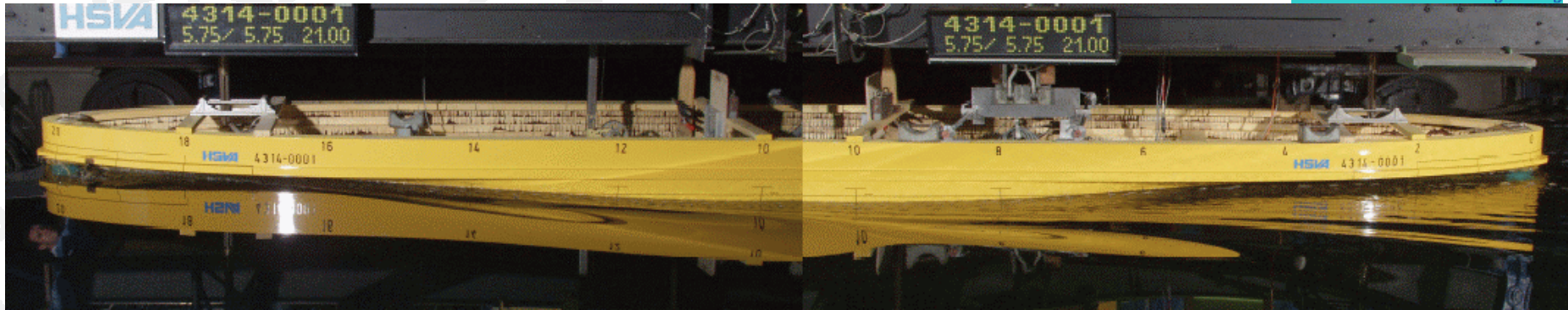
## CAD- Model



## Flow Simulation

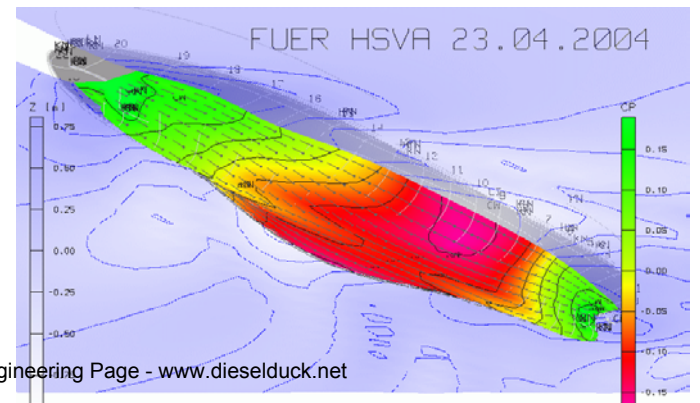


# Model Tests HSVA



**HSVA- Tests confirm Concept, CFD- Calculations and performance.**

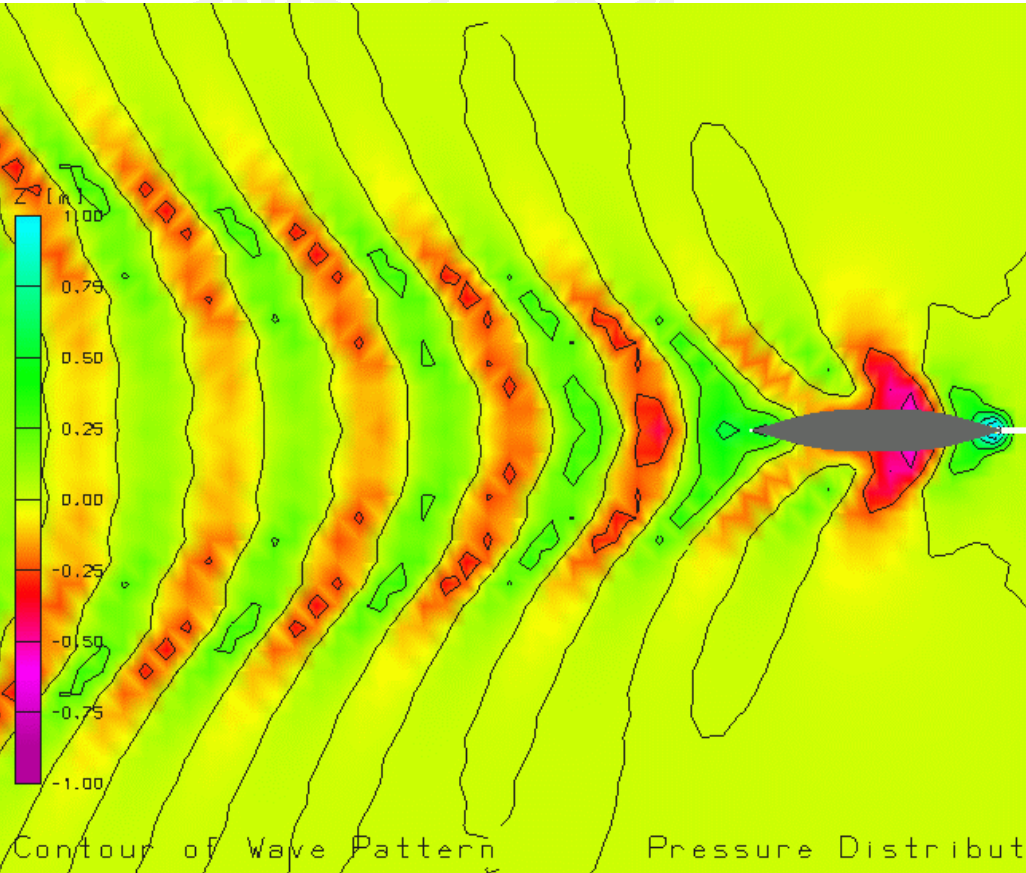
**Problem: Scaling Laws, Feathering- position of Bow Prop.**



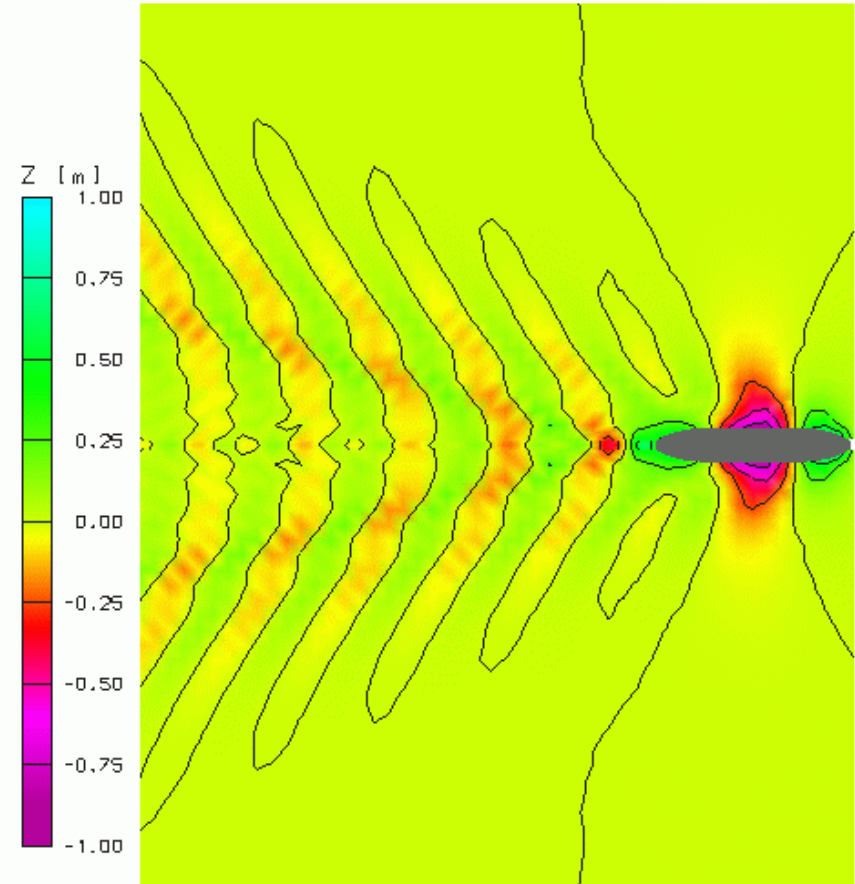


# BCF Wake Wash Requirements

Existing C-Class, 21 knots, 5850 t



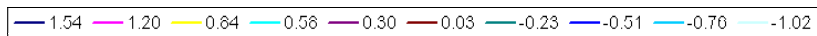
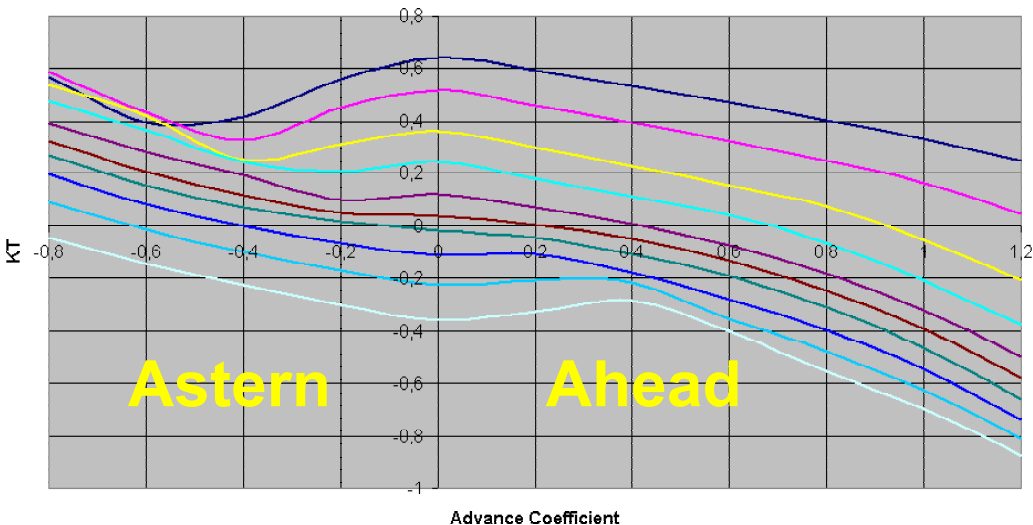
Our Design, 21 knots, 9600t



**Comparison shows TC – requirement clearly met.**

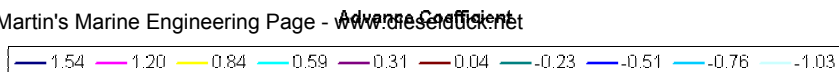
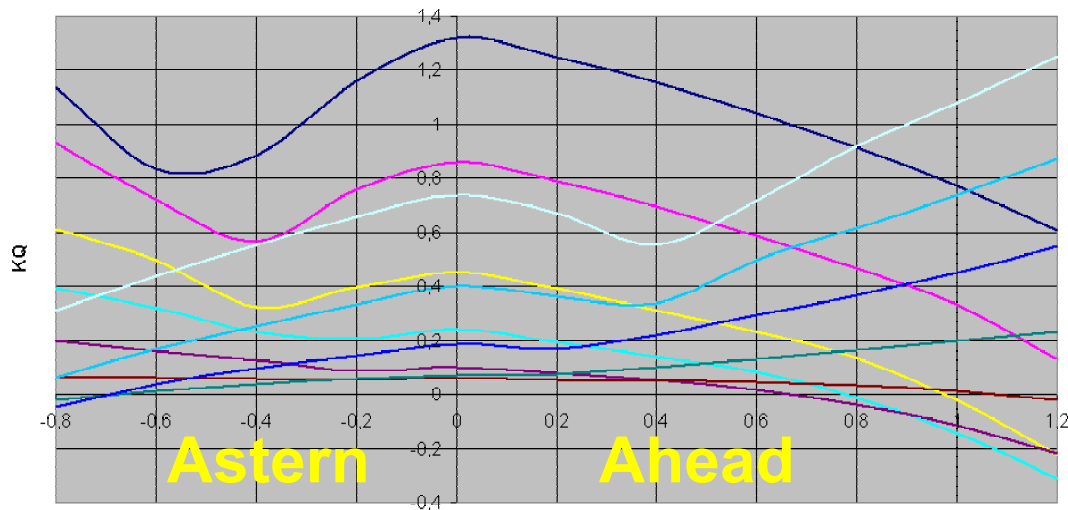
# Preparing Manoeuvring Simulations

KT Curves CPP



**Stock- Prop. (WPM/Nörönnä) tested in HSVA-Cav. Tunnel in 2 Quadrants and 10 Pitches.**

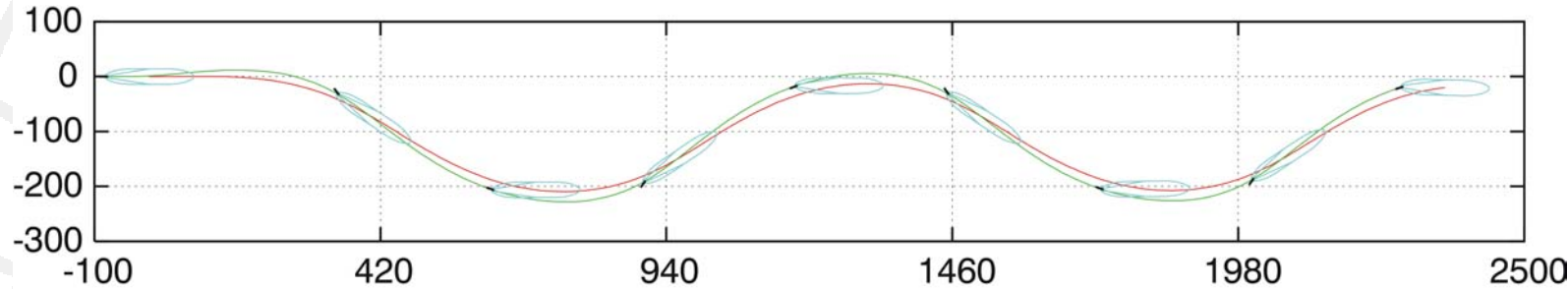
KQ Curves CPP



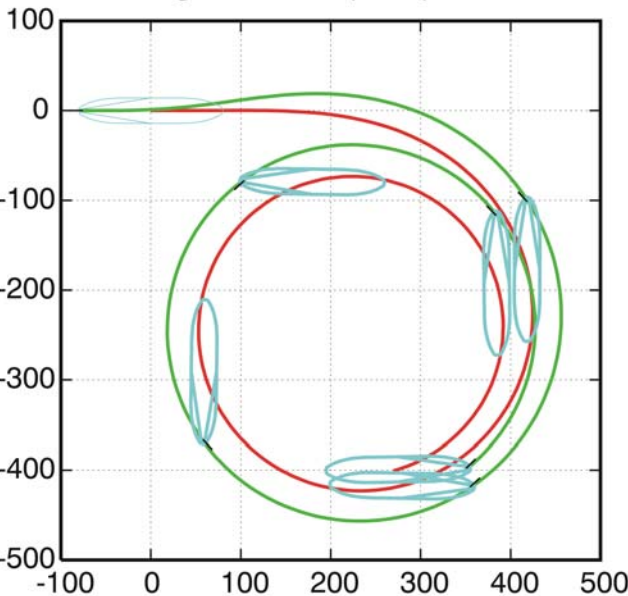
**Results used to calibrate TUHH lifting line model.**

# Manoeuvring (1)

## 20/20ZIGZAG XB 1 pump DESIGN



## 44 Deg. TC XB 1 pump DESIGN



Track of COG ———  
Track of GPS ———

- 1.Oversh 13.5 Deg.: t= 41 s,v= 19.8 kn ———
- Zero Heading No. 1 t= 73 s,v= 20.4 kn ———
- 2.Oversh 14.2 Deg.: t= 101 s,v= 18.7 kn ———
- Zero Heading No. 2 t= 134 s,v= 19.6 kn ———
- 3.Oversh 13.8 Deg.: t= 163 s,v= 18.2 kn ———
- Zero Heading No. 3 t= 197 s,v= 19.4 kn ———
- 4.Oversh 13.8 Deg.: t= 226 s,v= 18.2 kn ———
- Zero Heading No. 4 t= 259 s,v= 19.3 kn ———

**Manoeuvring Simulations confirm turning rate and overshoot.**

**Building- Spec. accepted.**

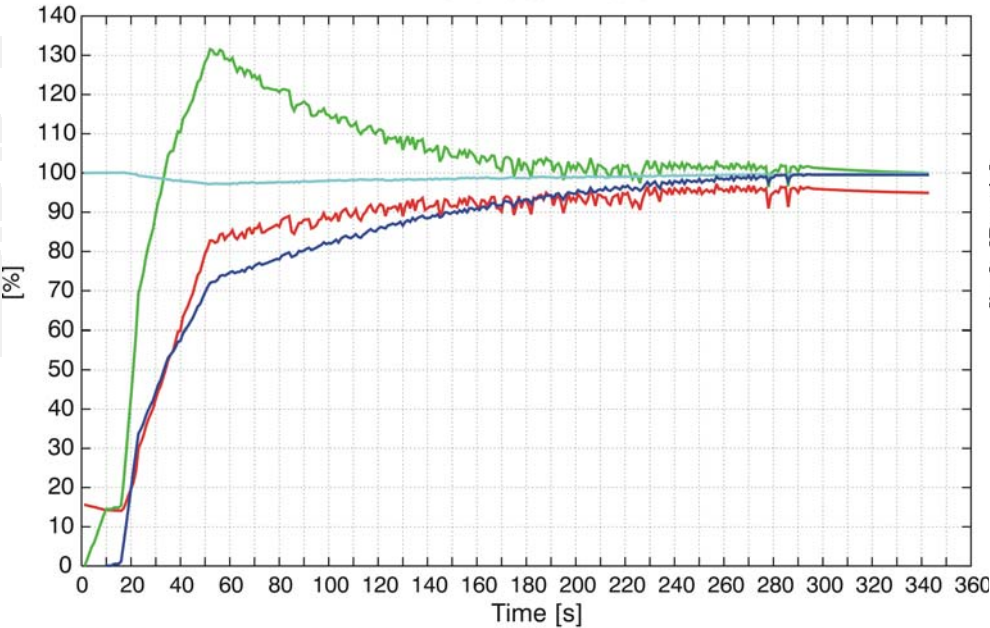
COG:Advance = 418 m,Tact. Diam= 417 m ———  
GPS:Advance = 495 m,Tact. Diam= 418 m ———

- 90 Deg. CoH : t= 51 s, v= 18.9 kn
- 180 Deg. CoH : t= 92 s, v= 15.6 kn
- 270 Deg. CoH : t= 136 s, v= 14.1 kn
- 360 Deg. CoH : t= 182 s, v= 13.4 kn
- 450 Deg. CoH : t= 228 s, v= 13.1 kn
- 540 Deg. CoH : t= 275 s, v= 13.0 kn

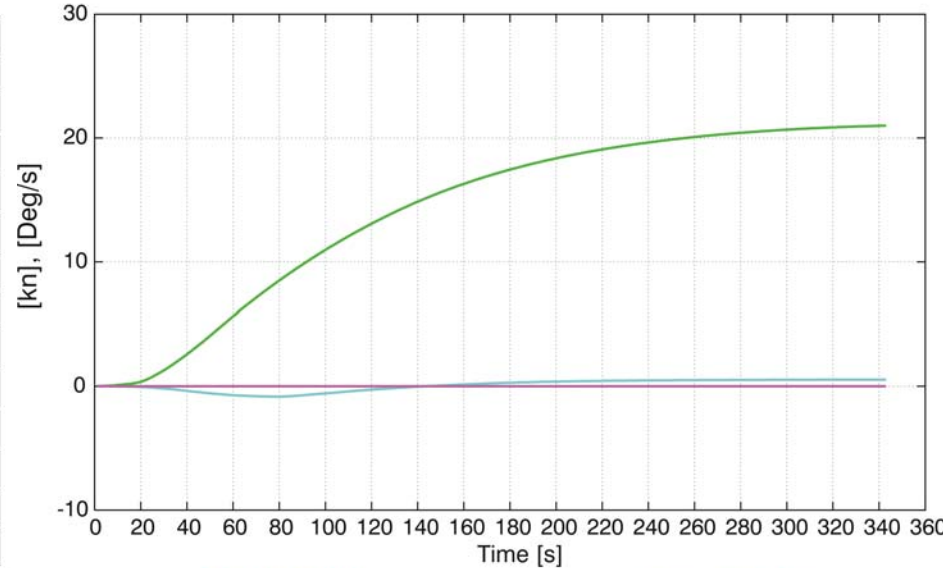
# Manoeuvring (2)

## Acceleration Manoeuvre w. WPM- CPP, 4 MaK 8M32, STN Drive Motors and Propulsion Control System

0 to 100%VDESIGN

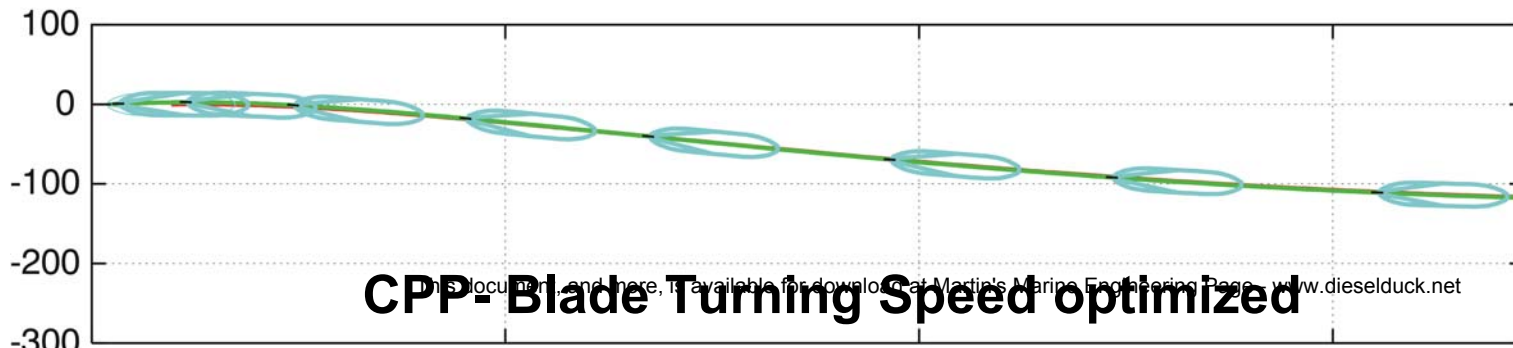


0 to 100%VDESIGN



Speed of COG —  
Speed of GPS —  
Turning Rate \*10 —  
Rudder Cross Flow —

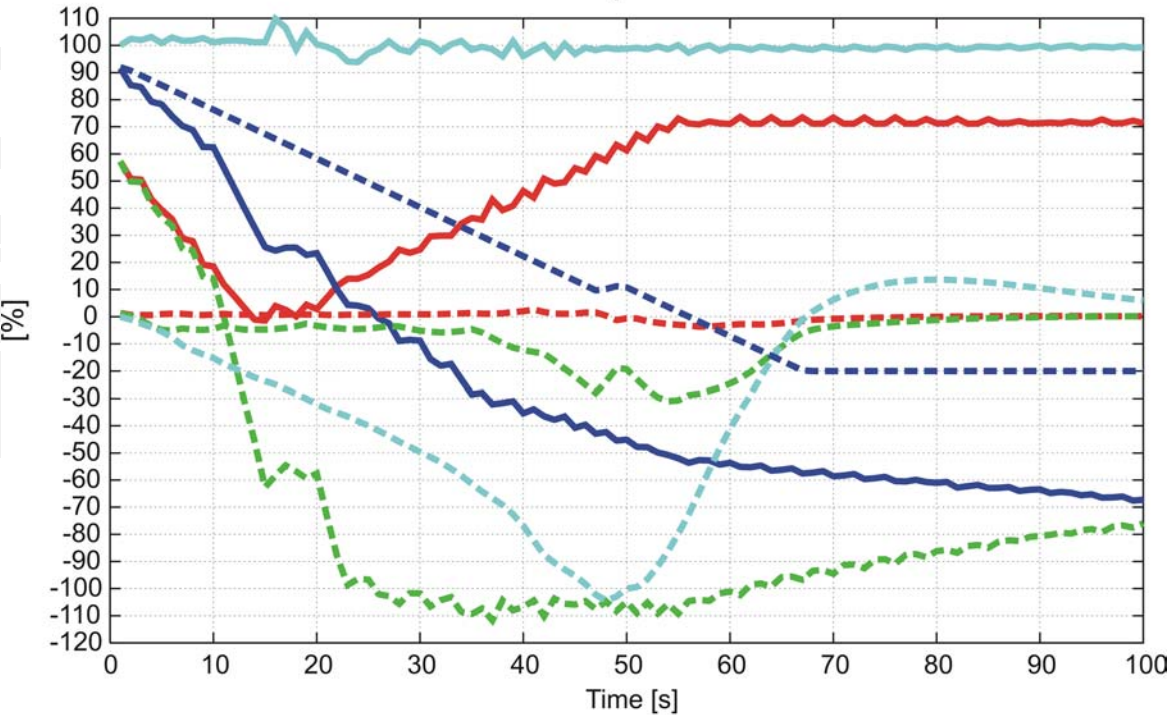
Torque — Thrust — Rev — Pitch —  
0 to 100%VDESIGN



CPP- Blade Turning Speed optimized

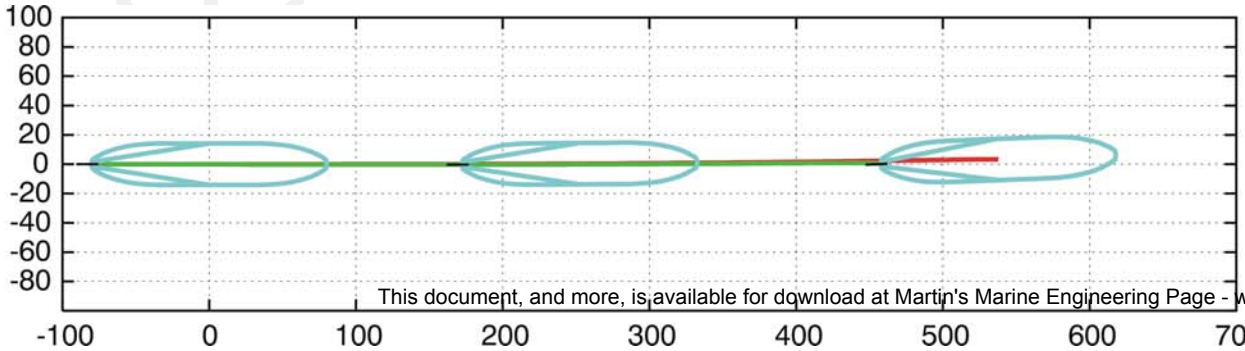
# Manoeuvring (3)

## Crash-Stop Manoeuvre w. WPM- CPP, 4 MaK 8M32, STN Drive Motor, PCS and defeathered Bow Propeller Crash Stop DESIGN



**Feathering- Concept and Automation designed acc. To Simulation Stopping Distance OK.**

— Torque PS    — Rev PS    - - - Torque BOW    - - - Rev BOW  
- - - Thrust PS    — Pitch PS    - - - Thrust BOW    - - - Pitch BOW in °



Pitch CMD: -107 % ,Speed: 21.0 kn, t=0  
 Zero Pitch: t: 26 s ,Speed: 17.0 kn  
 Stopping Distance: 537 m, t= 100 s  
 Transv. Distance: -3 m

— Track of C.O.G.  
— Track of GPS

# Active Pass- Problem

**Active Pass:**

**extreme Current  
up to 8 kn**

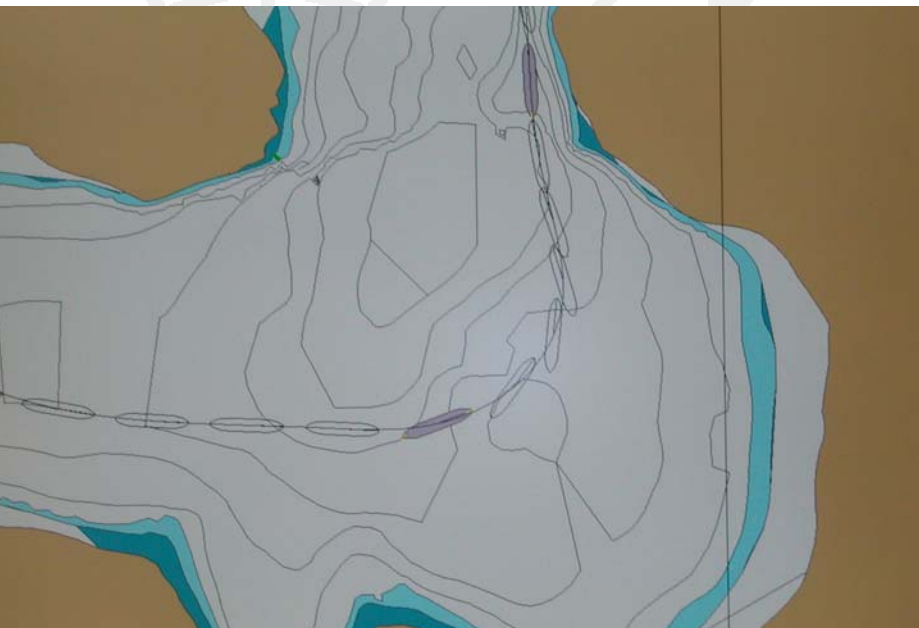
**Difficult waters, many  
rocks**

**Tranport- Canada  
requirement**



**Active Pass- Ability clear requirement of BCF.**

**How to demonstrate it ???**



**Virtual Testing of Active Pass at DMI /Lyngby.**

**TUHH/FSG- Manoeuvring model transferred to DMI- SIMFLEX.**

**Both Models agree in manoeuvring performance.**

**Active Pass shows no problems.**

**BCF accepts performance.**



# Conclusions

- 1. The new concept is competitive.**
- 2. New concepts require detailed investigations based on first principle design methods**
- 3. New concepts require multi disciplinary engineering on a high level**
- 4. Nothing is more practical than a good theory !!**